

Effects of Air Temperature, Mean Radiant Temperature, and Air Velocity on the Globe and Operative Temperatures

Mohammad Kazkaz, Mousa Sattouf

Abstract—In this paper is presented a theoretical comparison between the globe and operative temperature, to determine if it is possible to use the operative temperature instead of globe temperature for appraisal of the thermal comfort, for that matter each of the relation of globe temperature, the relation of operative temperature, the relation of the difference between operative and globe temperature as function of air velocity, mean radiant temperature and air temperature are presented.

Keywords—Thermal environment, thermal comfort, globe temperature, operative temperature.

I. INTRODUCTION

Thermal comfort is defined as the condition of mind in which satisfaction with the thermal environment is expressed. The definition is easy to understand but it is hard to express it by mathematical relation, because it is required to take into account number of environmental and personal parameters. Air temperature, radiant temperature, humidity and air velocity are the four basic environmental parameters which define the environmental thermal state. Together with the metabolic heat rate generated by human body activity and clothing worn by a person, they supply the six fundamental parameters that define human thermal environments [5]. Several instruments have been designed which give the effects of the six fundamental parameters on the thermal state of environment. The globe thermometer is one of these instruments. In this paper is shown the possibility of replacing the globe temperature instead of operative temperature for assessment thermal state of environment.

II. THERMAL COMFORT

The Thermal comfort has been defined by The American

This work was supported in part by the Energy Institute, Faculty of Mechanical Engineering, Brno University of Technology, under the projects FSI-S-11-6 and GAČR 101/09/H050.

M. Kazkaz is with Energy Institute, Faculty of Mechanical Engineering, Brno University of Technology BUT, Brno, Czech republic (phone: 420-541-143244; fax: 420-541-143345; e-mail: y138898@stud.fme.vutbr.cz).

M. Sattouf is with Department of Power Engineering, Faculty of Electrical Engineering and Communications, Brno University of Technology BUT, Brno, Czech republic (phone: 420-541-146756; fax: 420-541-146210; e-mail: xsatto00@stud.feec.vutbr.cz).

Society of Heating, Refrigerating and Air-Conditioning Engineers (ASHRAE) as the condition of the mind in which satisfaction is expressed with the thermal environment'' [1]. The thermal comfort describes a person's psychological case of mind and is usually referred in terms of whether someone is feeling too hot or too cold. When it is needed to determinate what will make a person feels comfortable, then we need to take into account a number of environmental and individual parameters, which make the definition of thermal comfort very difficult.

So for simplifying evaluation of the thermal comfort, indexes and derived quantities are used, including the combined effect of many or all of the factors determining the thermal state of the environment, including for example; The globe temperature, the operative temperature (temperature measured by globe thermometer), equivalent temperature, index PMV (Predicted mean vote), index PPD (predicted percentage dissatisfied) and index DR (draught rate). In this article the globe temperature and operative temperature are mainly discussed to evaluate the thermal state of the environment.

III. GLOBE TEMPERATURE

The globe temperature is quantity that measured directly by globe thermometer. The globe thermometer is one of the most common mean radiant temperature measurement devices. the globe thermometer consists of a blank copper sphere of diameter 150 mm (or 100 mm), coated with matt black color, the blank copper sphere contains an ordinary thermometer with its bulb fixed at the center of the sphere, without source of heat. In steady state the globe thermometer reaches the thermal equilibrium when radiant heat flux from the environment into the sphere is in balance with the convective heat flux from the surface of sphere to the environment[9][10].

$$q_c = q_r \quad (1)$$

By Stefan – Boltzmann's law, the radiation gain may be expressed by the equation:

$$q_r = \sigma \cdot \varepsilon (T_r^4 - T_g^4) \quad (2)$$

Where:

ε is a numerical constant depending on the emissivity of the sphere, for a surface painted matt black the value of ε is about 0.95 .

$\sigma = 5.67 \times 10^{-8}$ is Stefan - Boltzmann's constant ($\text{W}/\text{m}^2 \cdot \text{K}^4$).

T_r is mean radiant temperature of the surrounding surfaces (K).

T_g is temperature of the sphere surface (K).

The convection heat transfer between the environment and the globe thermometer is given by (3):

$$q_c = \alpha_{cg} (T_g - T_a) \quad (3)$$

Where:

T_a is air temperature (K).

α_{cg} is the convection heat transfer coefficient between air and the globe thermometer ($\text{W}/\text{m}^2 \cdot \text{K}$).

For natural heat flow is:[4]

$$\alpha_{cg} = 1.4 \left(\frac{|t_a - t_g|}{D} \right)^{0.25} \quad (4)$$

In case of forced heat flow; according to ISO 7726 is: [4]

$$\alpha_{cg} = 6.3 \frac{w^{0.6}}{D^{0.4}} \quad (5)$$

Where:

t_a is air temperature ($^{\circ}\text{C}$).

t_g is temperature of the sphere surface ($^{\circ}\text{C}$).

D is diameter of sphere (m).

w is air velocity (m/s).

The thermal equilibrium equation is written as following:

$$\alpha_{cg} (T_g - T_a) = \sigma \cdot \varepsilon (T_r^4 - T_g^4) \quad (6)$$

In this research the forced convection has been used because it is easier to express the globe temperature as a function of air velocity. Then the globe temperature can be written as following:

$$6.3 \frac{w^{0.6}}{D^{0.4}} T_g + \sigma \cdot \varepsilon T_g^4 = 6.3 \frac{w^{0.6}}{D^{0.4}} T_a + \sigma \cdot \varepsilon T_r^4 \quad (7)$$

For globe thermometer the following conditions are applied: Diameter of sphere $D=0.15$ m. All physical properties are constant, emissivity of sphere $\varepsilon=0.95$ and Stefan - Boltzmann's constant is 5.67×10^{-8} ($\text{W}/\text{m}^2 \cdot \text{K}^4$), then:

$$13.46 \cdot w^{0.6} \cdot T_g + 5.3865 \cdot 10^{-8} \cdot T_g^4 = 13.46 \cdot w^{0.6} \cdot T_a + 5.3865 \cdot 10^{-8} \cdot T_r^4 \quad (8)$$

From last equation it is difficult to write the globe temperature as function of air velocity, mean radiant and air temperature, because it is transcendental equation, which requires an iterative solution. In this article the Mathcad program was used to solve this equation and found the globe temperature as function of other variables as (9):

$$t_g = f(w, t_r, t_a) \quad (9)$$

Fig.1 and Fig.2 show the globe temperature as function of velocity, mean radiant temperature and air temperature (two viewpoints). Where these figures show three surfaces, each one represents the relation between globe temperature and air velocity for a constant air temperature.

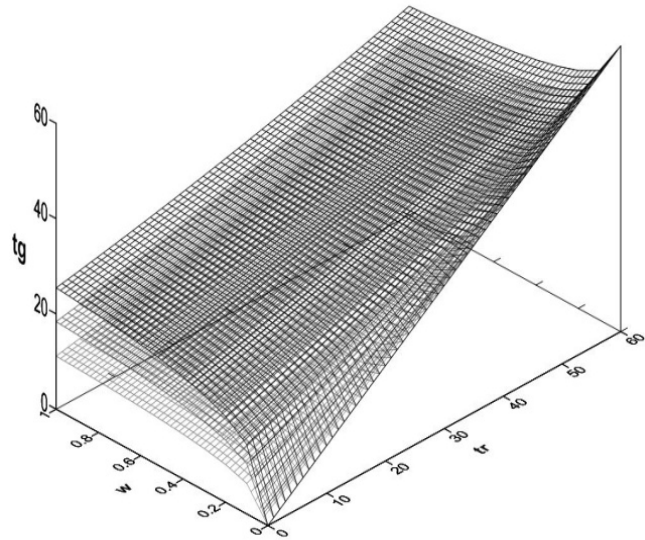


Fig.1 Globe temperature for $t_a=15^{\circ}, 25^{\circ}, 35^{\circ}$ (right side view)

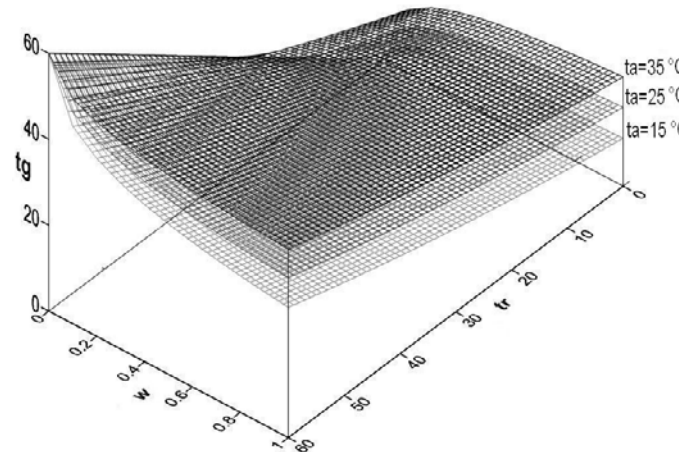


Fig.2 Globe temperature for $t_a=15^{\circ}, 25^{\circ}, 35^{\circ}$ (left side view)

IV. OPERATIVE TEMPERATURE

The operative temperature (t_o) is a uniform temperature of a radiantly black enclosure in which an occupant would exchange the same amount of heat by radiation and convection as in the actual non uniform environment. According to ČSN EN ISO 7726 [2] is:

$$t_o = t_a + (1-A)(t_r - t_a) \quad (10)$$

There are many methods for calculating the operative temperature. For example, it can be calculated according to ČSN EN ISO 7730 [3] where:

$$A = \frac{\alpha_c}{\alpha_c + \alpha_r} \quad (11)$$

Where:

α_c, α_r [$\text{W}/\text{m}^2 \cdot \text{K}$] are the coefficients of heat transfer by convection and radiation, respectively, on the body surface t_a .

t_r [°C] is air temperature and mean radiant temperature. For low air velocity $w < 0.2$ m/s it is possible to replace operative temperature with globe temperature [7].

$$t_g = t_o, T_g = T_o \tag{12}$$

Where:

T_g, t_g is globe temperature in unite of [K], [°C] respectively.

T_o, t_o is operative temperature in unite of [K], [°C] respectively.

Other reference [2], considers for low air velocity $w < 0.2$ m/s and small difference of temperature $|t_a - t_r|$, it is possible assess the operative temperature as arithmetic mean of air and mean radiant temperature.

$$t_o = \frac{t_a + t_r}{2} \tag{13}$$

Table 1 shows value of coefficient A depending on the air velocity w.

Table.I Dependence the coefficient A on the air velocity w

w [m/s]	< 0.2	0.2 to 0.6			> 0.6 to 1	
A [2]	0.5	0.6			0.7	
w [m/s]	< 0.2	0.3	0.4	w [m/s]	< 0.2	0.3

It is possible to calculate the coefficient A from the following equation [2]:

$$A = 0.73 \cdot w^{0.2} \tag{14}$$

From the equations (10) and (14) can be found the operative temperature as function of air velocity, mean radiant temperature and air temperature as:

$$t_o = f(w, t_r, t_a) \tag{15}$$

Fig.3 and Fig.4 show the relation of operative temperature with air velocity, mean radiant temperature and air temperature (two viewpoints).

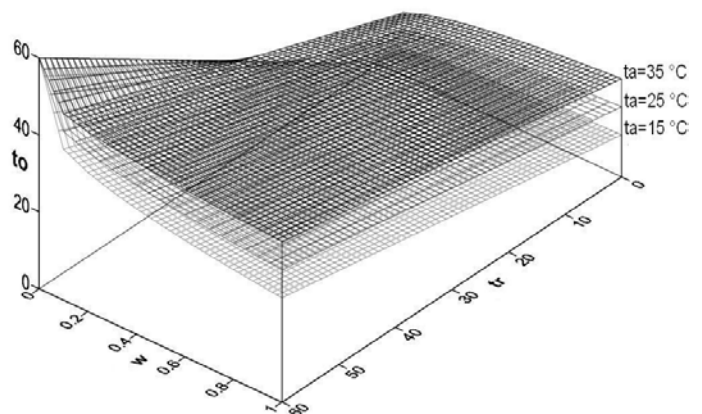


Fig. 3 Operative temperature for $t_a=15^\circ\text{C}, 25^\circ\text{C}, 35^\circ\text{C}$ (left side view)

These figures show three surfaces, each one represents the relation between operative temperature and air velocity for a constant air temperature.

The relation between operative temperature and globe temperature is given by similarity between the heat transfer on the surface of human body and the heat transfer on the surface of globe thermometer, and because of the globe temperature is easily obtained by reading it directly from the globe thermometer [8].

Fig.5 and Fig.6 show a comparison between the operative temperature and globe temperature, where Fig. 3b is for range of mean radiant temperature $40^\circ\text{C} \geq t_r \geq 0^\circ\text{C}$ and air velocity in the rang $1 \text{ [m/s]} \geq w \geq 0.2 \text{ [m/s]}$.

The figures show two surfaces, each one represents the relation between the difference of operative and globe temperature and air velocity for a constant air temperature.

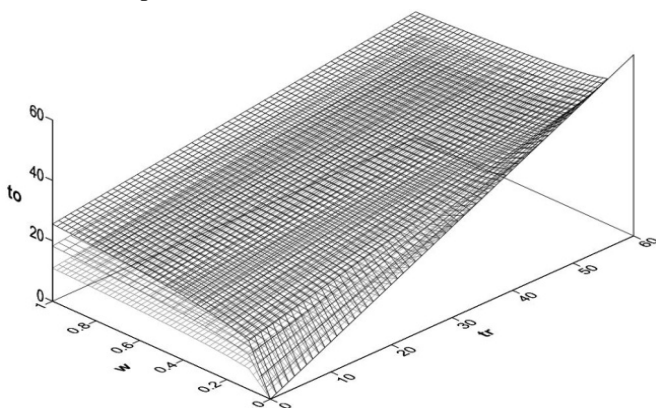


Fig. 3 Operative temperature for $t_a=15^\circ\text{C}, 25^\circ\text{C}, 35^\circ\text{C}$ (right side view)

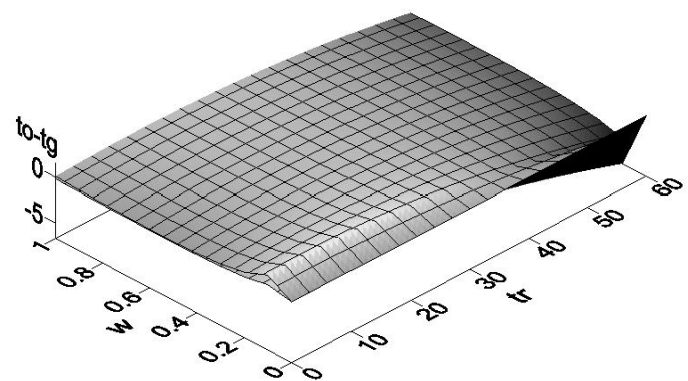


Fig.5 Difference of operative and globe temperature for $t_a=15^\circ, 35^\circ$ (sidelong view)

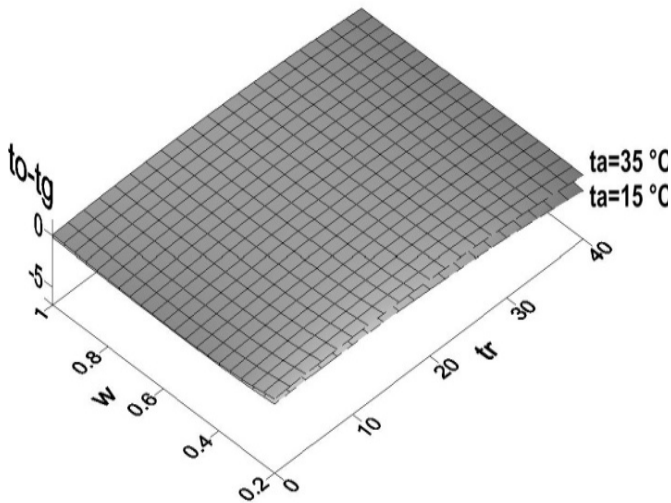


Fig.6 Difference of operative and globe temperature for $t_a=15^\circ\text{C}, 35^\circ\text{C}$ (upper view)

V. DISCUSSION

According to obtained results the difference between the operative temperature and globe temperature depends on the air velocity and further on the difference between the mean radiant temperature and air temperature, where:

- For the difference $|t_r - t_a| \leq 10$ K and air velocity greater than 0.2 m/s, then the difference $|t_o - t_g|$ is between 0 K and 0.9K.
- For air velocity less than 0.2 m/s, then the difference between the globe and operative temperature is up to 1.35K.

The best result is for $|t_r - t_a| \leq 5$ K, and air velocity smaller than 0.2[m/s], where in this range the difference between operative temperature and globe temperature is compatible with the Czech standards [4], where the accuracy in the globe temperature measurement is $\pm 0.5^\circ\text{C}$ for air temperature between 20°C to 50°C .

Fig.7, Fig.8 and Fig.9 show the difference between operative and globe temperature for constant mean radiant and air temperature.

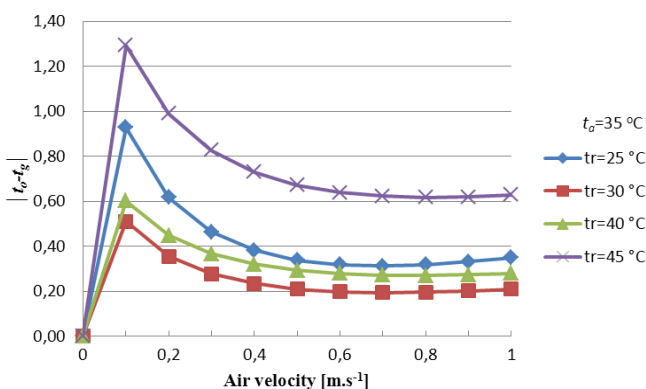


Fig.7 Difference between operative and globe temperature as function of air velocity for $t_a=35^\circ\text{C}$

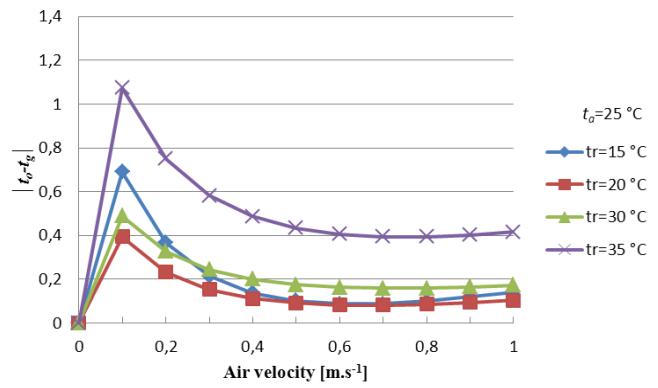


Fig.8 Difference between operative and globe temperature as function of air velocity for $t_a=25^\circ\text{C}$

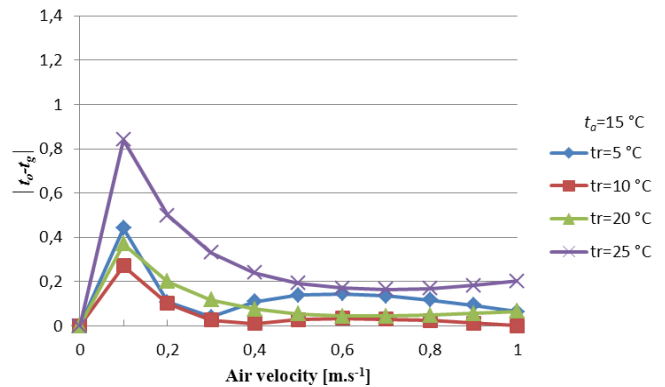


Fig.9 Difference between operative and globe temperature as function of air velocity for $t_a=15^\circ\text{C}$

Fig.10, Fig.11 and Fig.12 show the difference between operative and globe temperature for constant mean radiant and air temperature for air velocity greater than 0.2 m/s.

The figures show two surfaces, each one represents the relation between the difference of operative and globe temperature and air velocity for a constant air temperature.

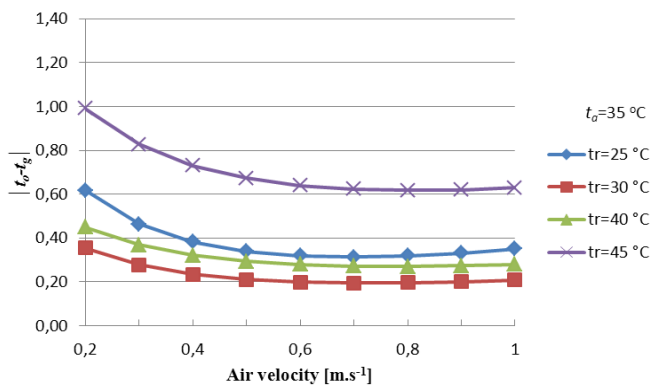


Fig.10 Difference between operative and globe temperature as function of air velocity (>0.2 m/s) for $t_a=35^\circ\text{C}$

VI. CONCLUSION

Depending on the previous discussion, it is possible to use the globe temperature instead of the operative temperature for air velocity $0.2 \geq w \geq 1$ m/s and the difference $|tr-t_a| \leq 5$ K to assessment the thermal state of environment. prior conclusion disagrees with the Statute-book [7], where is written that for air velocity less than 0.2 m/s, operative temperature can be replaced by the globe temperature. Another reference, the Society of Environmental Engineering [5], they had the same conclusion like in this article. By analyze the results it is possible to say that the difference between the operative temperature and globe temperature in range of velocity of $0 < w < 0.2$ m/s is due to omit the effect of free convection in this range.

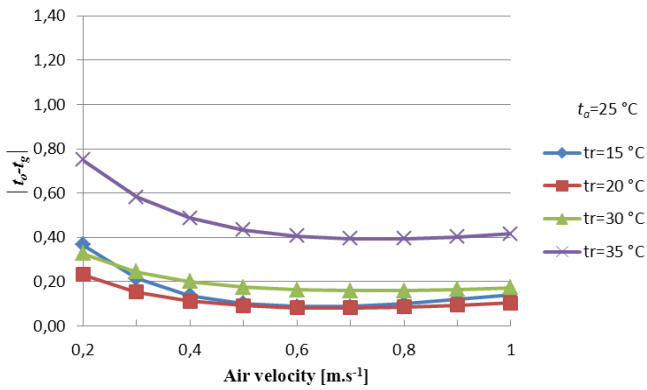


Fig.11 Difference between operative and globe temperature as function of air velocity(>0.2 m/s) for $t_a=25^\circ\text{C}$

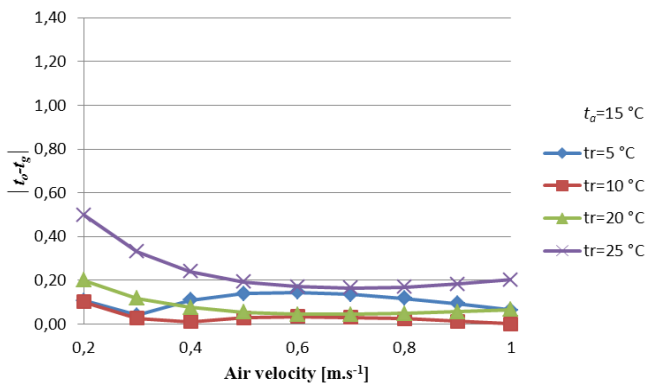


Fig.12 Difference between operative and globe temperature as function of air velocity(>0.2 m/s) for $t_a=15^\circ\text{C}$

APPENDIX

Table. II Characteristics of measuring instruments

Class S (thermal load)						Notice
	quantity	symbol	measurement range	accuracy	stabilization time (90 %)	

Basic quantities	Air velocity					Parameter when studying the issue of comfort. It is recommended that was expressed as a standard deviation of the speed. In cold environments, it is recommended that was used the device class C for any type of the analysis (comfort or extreme thermal load)
	absolute humidity	p_a	0.5 to 6 kPa	± 0.15 kPa This value must be guaranteed for air and walls temperature equal to or greater than 30 °C and for the difference ($t_r - t_a$) at least 10 °C	Soon as possible. value is determined as a characteristic of the measuring device	
Derived quantities	Temperature of wet thermometer	t_{nw}	5 to 40 °C	± 0.5 °C	Value is determined as a characteristic of the measuring device	The sensor characteristics is prescribed
	Globe temperature	t_g	20 to 120 °C	20 to 50 °C: ± 0.5 °C >50 to 120 °C: ± 0.5 °C	The same	The sensor characteristic is prescribed. Temperature of the globe can be used to estimate the mean radiant temperature in cold, slight and hot zone.
	Wet Globe temperature	t_{wg}	0 to 80 °C	± 0.5 °C	The same	Accuracy of measurement for globe temperature to determine t need not be the same as the accuracy for measurement of globe temperature, as it derived values. The sensor characteristic is prescribed.

ACKNOWLEDGMENT

Authors would like to express their deepest appreciation to all those who provided them the possibility to complete this work. A special gratitude they give to Prof. Ing. Melan Pavelek, CSc. for the continuous support of the research.

REFERENCES

- [1] *Thermal Environment Conditions for Human Occupancy*, ANSI/ASHRAE Standard 55-2004.
- [2] *Moderate thermal environments - Determination of the PMV and PPD indices and specification of the conditions for thermal comfort*, ČSN EN ISO 7730, Prague, 1997.
- [3] *Tepelného prostředí - Analytické stanovení a interpretace tepelného komfortu pomocí výpočtu ukazatelů PMV a PPD a kritéria místního tepelného komfortu*, ČSN EN ISO 7730, Praha, Český normalizační institut., 2005.
- [4] *Thermal environments: Instruments and methods for measuring physical quantities*, ČSN ISO 7726, Prague: Czech Standards Institute, 1993.
- [5] Directive Society of Environmental Engineering 0S 01: *Operative temperature in practice*. 3rd.ed. Prague: 2010. (original in Czech) - Směrnice Společnost pro Techniku Prostředí - 0S 01: *Operativní teplota v praxi*. 3. vyd. Praha: 2010.
- [6] K. PARSONS, *Human thermal environments: the effects of hot, moderate, and cold environments on human health, comfort, and performance*, 2nd ed. London: Taylor, 2003, pp 527. ISBN 04-152-3793-9.
- [7] Statute-book No. 523/2002 Sb., CZ, Year 2002, (original in Czech).
- [8] M. KAZKAZ, J. WEISMANOVÁ, "ASSESSMENT OF THE THERMAL COMFORT: THE OPERATIVE TEMPERATURE," in *Mezinárodní Masarykovy konf. pro doktorandy a mladé vědecké*

pracovníky, Hradec Králové, 2012, pp. 2870-2877. ISBN: 978-80-905243-3-0.

- [9] M. KAZKAZ, M. PAVELEK, "Operative temperature and globe temperature," in *31. mezinárodní konference Setkání kateder mechaniky tekutin a termomechaniky*. Brno, Vysoké učení technické v Brně, 2012, pp 89-92, ISBN: 978-80-214-4529- 1.
- [10] H. M. VERNON, "The measurement of radiant heat in relation to human comfort," *J. Physiol.* 70, Proc. 15, 1930.



Ing. Mohammad Kazkaz, PhD student in Energy Institute, Faculty of Mechanical Engineering, Brno University of Technology, Brno, Czech Republic. He graduated from Energy Department, Faculty of mechanical Engineering, Aleppo University, Aleppo, Syria in 2006. He worked as assistant teacher in Mechanical Energy Department, faculty of mechanical and Electrical Engineering, Albaath

University, Homs, Syria in 2008.



Ing. Mousa Sattouf, PhD student in power electrical engineering department, Faculty of electrical engineering and communications, Brno university of technology, Brno, Czech republic..He graduated from Power Electrical Department, Faculty of Mechanical and Electrical Engineering, Damascus University, Damascus, Syria in 2006. He worked a Lecturer and teacher in laboratories of the department, from which

he graduated in 2007, and worked as assistant teacher in Electrical Drive Department, Faculty of Electrical Engineering, Aleppo University, Aleppo, Syria in 2008.