# Wiener-Hammerstein Models Identification 

A. Brouri ${ }^{*}$,


#### Abstract

This work discusses the identification of nonlinear systems structured in blocks. Presently, the proposed method is addressed to Wiener-Hammerstein models. Hammerstein and Wiener models are nonlinear representations of systems composed by connecting of a nonlinearity element $f($.) and a linear subsystem $G(s)$ in the form $f()-.G(s)$ and $G(s)-f($.$) respectively. The$ identification of nonlinearity blocks and linear subsystems is not a trivial problem, and has attracted a lot of research interest. The linear subsystems $G_{i}(s)$ and $G_{o}(s)$ are allowed to be nonparametric and of unknown structure. Presently, the system nonlinearity is static and may be noninvertible. Moreover, this latter is of unknown structure and is only supposed to be well approximated, within any subinterval belonging to the working interval, with a polynomial of unknown order and parameters. Then, using a frequency identification method, a two-phase algorithm is presented for identifying the linear subsystems $G_{i}(s)$ and $G_{o}(s)$ (the frequency complex gains) and the nonlinearity element $f($.$) .$

The procedure is illustrated with simulated and experimental data. The proposed strategy involves simples input signals.


Key-Words-Block-oriented models, Nonlinear systems, Wiener systems, Hammerstein systems, Frequency identification, nonparametric systems.

## I. Introduction

Wiener-Hammerstein systems consist of a series connection including a nonlinear static element sandwiched with two linear subsystems (Fig. 1). Clearly, this model structure is a generalization of Hammerstein and Wiener models and so it is expected to feature a superior modelling capability. This has been confirmed by several practical applications e.g. paralyzed skeletal muscle dynamics (Bai et al., 2009). As a matter of fact, Wiener-Hammerstein (WH) systems are more difficult to identify than the simpler Hammerstein and Wiener models. The complexity of the former lies in the fact that these systems involve two internal signals not accessible to measurements, whereas the latter only involve one. Then, it is not surprising that only a few methods are available that deal with WH system identification. The available methods have been developed following three main approaches i.e. iterative nonlinear optimization procedures (e.g. Marconato et al., 2012), stochastic methods (Bershad et al., 2001; Pillonetto et al., 2011); frequency methods (Giri et al., 2013; Brouri et al., 2014). The proposed identification methods also differ by the type of assumptions, made on both the system dynamics and the input signals, and the nature of convergence analysis results. Roughly, the iterative methods necessitate a large amount of data, since computation time and memory usage drastically increase, and have local convergence properties

[^0]which necessitates that a fairly accurate parameter estimates are available to initialize the search process. This prior knowledge is not required in stochastic methods but these are generally relied on specific assumption on the input signals (e.g. gaussianity, persistent excitation....) and on system model (e.g. MA linear subsystems, smooth nonlinearity). The frequency methods are generally applied to nonparametric systems under minimal assumptions and only require simple periodic excitations. But, they sometime necessitate several data generation experiments.

In this paper, the problem of identifying WH systems is addressed, for simplicity, in the continuous-time. Unlike many previous works, the model structure of the two linear subsystems is entirely unknown. Furthermore, the static nonlinearity is also of unknown structure and is not required to be invertible. This is only supposed to be well approximated, within any subinterval belonging to the working interval, with a polynomial of unknown order and parameters. The order $p$ and the parameters of the polynomial can vary from one subinterval to another. It turns out that the complexity of the identification problem lies in: (i) the fact that the internal signals $u_{i}$ and $u_{o}$ are not accessible to measurement (Fig. 1); (ii) the nonparametric and nonlinear nature of the system. Given the system nonparametric nature, the identification problem is presently dealt with by developing a two-stage frequency identification method, involving periodic inputs. First, a set of points of the nonlinearity is identified using simple experiments; the size of this set is arbitrarily chosen by the user. Then, the frequency responses of the two linear subsystems are estimated for a number of frequencies; in turn, this number can be made arbitrarily large. The frequency gain estimator design relies on input/output Fourier series expansions. Compared to previous works, all involved estimators are presently shown to be consistent. Furthermore, the input excitation signals are deterministic and no particular assumption is made on the external noise except for stationarity.

This paper is organized as follows: the identification problem is formally described in Section 2; then, the identification of the nonlinearity is coped with in Section 3; the identification of the linear subsystems is dealt with in Section 4. Simulation examples are provided in Section 5 to illustrate the performances of the whole identification method.


Figure 1. Wiener-Hammerstein System Model.

## II. Identification Problem Statement

We are interested in systems that can be described by the Wiener-Hammerstein model of Fig. 1 where the different blocs are analytically described as follows

$$
\begin{gather*}
y=w+\xi,  \tag{1}\\
w=G_{o}(s) u_{o},  \tag{2}\\
u_{o}=f\left(u_{i}\right), u_{i}=G_{i}(s) v, \tag{3}
\end{gather*}
$$

where $G_{i}(s)$ and $G_{o}(s)$ are the transfer functions of the linear subsystems, $f($.$) denotes the static nonlinearity, and$ $\xi$ is an external noise. As, the signals $u_{i}, u_{o}, w$ and $\xi$ are not accessible to measurements, the identification procedure of the nonlinear system must only relay on the external signals $v$ and $y$. The signal $\xi$ is supposed to be a zeromean stationary sequence of independent random variables and ergodic. The static nonlinear element $f($.) has any shape and, in particular, may be noninvertible. It is only assumed that $f($.$) is smooth so that it can be accurately represented,$ within any finite interval, with a polynomial of finite order. Of course the polynomial order depends on the interval length.

The transfer functions $G_{i}(s)$ and $G_{o}(s)$ are of unknown structures. There are only supposed to be asymptotically stable and with nonzero static gain (i.e. $G_{i}(0) \neq 0$ and $\left.G_{o}(0) \neq 0\right)$. System stability is coherent with open-loop system identification. Also, note that the nonzero static-gain requirement is satisfied by most real life systems. In fact, only derivative systems make an exception that can be coped with using ad-hoc adaptations of the method developed in this paper. The problem complexity also lies in the fact that the (unavailable) internal signals ( $u_{i}, u_{o}, w$ and $\xi$ ) are not uniquely defined from an input-output viewpoint. In effect, if $\left(G_{i}(s), f(x), G_{o}(s)\right)$ is representative of the system then, any model of the form $\left(G_{i}(s) / k_{1}, k_{2} f\left(k_{1} x\right), G_{o}(s) / k_{2}\right)$ is also representative whatever the real numbers $k_{1} \neq 0, k_{2} \neq 0$. Therefore, the fact that $G_{i}(0) \neq 0$ and $G_{o}(0) \neq 0$ implies that, without reducing the problem generality, one can assume $G_{i}(0)=G_{o}(0)=1$.

Remark 1. The present identification method is quite different of previous frequency methods. In (Bershad et al., 2001; Pillonetto et al., 2011) the identification methods require a special design of the input signal. In (Paduart et al., 2012) the identification method is based on the best linear approximation technique using class of Gaussian signals.

## III. Identification Of Static Nonlinearity

In this section, we want to treat the problem of identifying a set of points belonging to non-linearity. In Section 2 it was shown that, if $k_{1}$ and $k_{2}$ are any nonzero real numbers, then any model of the form $\left(G_{i}(s) / k_{1}, k_{2} f\left(k_{1} x\right), G_{o}(s) / k_{2}\right)$ is
representative of the system. Accordingly, the system to be identified is described by the transfer functions

$$
\begin{gather*}
\bar{G}_{i}(s)=G_{i}(s) / G_{i}(0), \bar{G}_{o}(s)=G_{o}(s) / G_{o}(0),  \tag{4a}\\
\bar{f}(x)=G_{o}(0) f\left(G_{i}(0) x\right), \tag{4b}
\end{gather*}
$$

Then, $\bar{G}_{i}(0)=\bar{G}_{o}(0)=1$. Under these conditions, if $v(t)$ is constant then the steady-state undisturbed output $w(t)$ depends only on the input value and the nonlinearity $\bar{f}($.$) .$ The number $n$ of points is arbitrary. Let $V_{1}=v_{m}<V_{2}<\cdots<V_{n}=v_{M}$ be the selected abscissas. To determine the points $\left(V_{j}, \bar{f}\left(V_{j}\right)\right)$, letting

$$
\begin{equation*}
v(t)=V_{j} \quad(t \geq 0) \text { for } j=1 \ldots n \tag{5}
\end{equation*}
$$

As the linear subsystem $G_{i}(s)$ is asymptotically stable, therefore the internal signal $u_{i}(t)$ is constant, and one has $u_{i}(t) \underset{t \rightarrow \infty}{\rightarrow} U_{i}^{j}$, then in the steady-state

$$
\begin{equation*}
U_{i}^{j}=G_{i}(0) V_{j} \text { for } j=1 \ldots n \text { and } t \geq N T_{r}, \tag{6}
\end{equation*}
$$

where $T_{r}$ should be comparable to the system rise time i.e. the time that is necessary for a system step response to reach $90 \%$ of its final value. Then, as the system is asymptotically stable, its step response settles down (i.e. gets very close to final value) after a transient period of $N T_{r}$ seconds with $N \geq 1$.

As the linear subsystem $G_{o}(s)$ is asymptotically stable, it follows that the steady-state of the internal signal $u_{o}(t)$ is constant i.e. $u_{o}(t) \underset{t \rightarrow \infty}{\rightarrow} U_{o}^{j}$, and is written for $t \geq N T_{r}$

$$
\begin{equation*}
U_{o}{ }^{j}=f\left(U_{i}^{j}\right)=f\left(G_{i}(0) V_{j}\right) \text { for } j=1 \ldots n, \tag{7}
\end{equation*}
$$

In which case, the undisturbed output $w(t)$ is also constant (in the steady-state) i.e. $w(t) \underset{t \rightarrow \infty}{\rightarrow} W_{j}$. It readily follows from (4b) and (7) that $W_{j}$ can be expressed as follows

$$
\begin{equation*}
W_{j}=G_{o}(0) U_{o}^{j}=\bar{f}\left(V_{j}\right) \text { for } j=1 \ldots n, \tag{8}
\end{equation*}
$$

Finally, notice that the steady-state undisturbed output $W_{j} \quad(j=1 \ldots n)$ can simply be estimated using the fact that $y(t)=w(t)+\xi(t)$ and $\xi(t)$ is zero-mean. Specifically, $W_{j}$ can be recovered by averaging $y(t)$ on a sufficiently large interval. Hence, a number of points of the nonlinear function $\bar{f}$ (.) can thus be accurately estimated by repeating the above experiment successively for $V_{1}$ to $V_{n}$.

These ideas are formalized in the two-stage identification procedure of Table 1.

TABLE I. NONLINEARITY IDENTIFICATION (NI)


The set of couples $\left(V_{j}, \hat{W}_{j}(N)\right)$, with $j=1 \ldots n$, is the number of different positions occupied by the working point. Note that each input value $V_{j}(j=1 \ldots n)$ is kept on during $N T_{r}$ seconds making possible for the output signal to settle down within each interval of the form $\left[(j-1) N T_{r} j N T_{r}\right]$ with $j=1 \ldots n$. Practically, one can determine a suitable value of $N$ by observing any step response of the system.

Proposition 1. The points of coordinates $\left(V_{j}, \hat{W}_{j}(N)\right)$, for $j=1 \ldots n$, obtained from the data collected on the time interval [ $0 n N T_{r}$ ], converge (in probability) to the trajectory of nonlinearity $\bar{f}($.$) as N \rightarrow \infty$.

Proof. From the asymptotic stability of linear subsystems $G_{i}(s)$ and $G_{o}(s)$, after the step Data acquisition, it follows from (5)-(8) that, in the steady-state, for each subinterval $\left[(j-1) N T_{r} j N T_{r}\right]$ with $j=1 \ldots n$, the working point $(v(t), w(t))$ moves on the path of nonlinearity $\bar{f}($.$) .$ Accordingly, for all values of input $V_{j}(j=1 \ldots n)$ the set of points $\left(V_{j}, w(t)\right)$, in the steady-state, occupy $n$ positions on the trajectory of nonlinearity $\bar{f}($.$) . On the other hand, using$ the fact that the noise $\xi$ in (1) is zero-mean

$$
\begin{align*}
& \lim _{N \rightarrow \infty} \hat{W}_{j}(N)=\lim _{N \rightarrow \infty} \frac{1}{N T_{r}} \int_{(j-1) N T_{r}}^{j N T T_{r}} y(t) d t \\
= & \lim _{N \rightarrow \infty} \frac{1}{N T_{r}} \int_{(j-1) N T_{r}}^{j N T_{r}} w(t) d t \text { for } j=1 \ldots n, \tag{11}
\end{align*}
$$

$$
\begin{equation*}
\lim _{N \rightarrow \infty} \hat{W}_{j}(N)=\lim _{N \rightarrow \infty} \frac{1}{N T_{r}} \int_{(j-1) N T_{r}}^{j N T_{r}} y(t) d t=\bar{f}\left(V_{j}\right), \tag{12}
\end{equation*}
$$

Finally we can conclude that

$$
\begin{equation*}
\lim _{N \rightarrow \infty}\left(V_{j}, \hat{W}_{j}(N)\right)=\left(V_{j}, \bar{f}\left(V_{j}\right)\right) \text { for } j=1 \ldots n \tag{13}
\end{equation*}
$$

which proves the Proposition 1.

## IV. Linear Subsystem Identification

In this section, an identification method is proposed to obtain estimates of the complex gain corresponding to the two linear subsystems $\bar{G}_{i}(s)$ and $\bar{G}_{o}(s)$ at the frequencies $k \omega(k=0,1, \ldots)$ whatever $\omega>0$. From the NI procedure (Table 1), one gets estimates of $n$ different points on the path of nonlinearity $\bar{f}($.$) . Furthermore, the larger the parameter$ $N$ is, the better the estimation accuracy. For simplicity, we presently suppose that the estimated points have been exactly determined.

Recall that, the static nonlinearity $f($.) can be accurately represented, within any finite interval, with a polynomial of finite order, where the polynomial order depends on the interval length.Then, one has for all $x \in\left[v_{m} v_{M}\right]$

$$
\begin{equation*}
f(x) \approx \sum_{l=0}^{p} c_{l} x^{l} \text { with } p<\infty, \tag{14a}
\end{equation*}
$$

Then, it is readily seen, using (4b) and (14a), that $\bar{f}($.$) can be$ developed as follows

$$
\begin{equation*}
\bar{f}(x) \approx \sum_{l=0}^{p} \bar{c}_{l} x^{l}, \tag{14b}
\end{equation*}
$$

with $\bar{c}_{l}=G_{o}(0) G_{i}(0)^{l} c_{l} \quad(l=0 \ldots p)$. All along this Section, the identified system is submitted to a given sine input

$$
\begin{equation*}
v(t)=x_{0}+V \sin (\omega t) \quad(\omega>0), \tag{15}
\end{equation*}
$$

where the amplitude $V>0$ and $x_{0}$ is any point in the working interval, this latter may be chosen equal to zero. Let $T$ be the corresponding period $(T=2 \pi / \omega)$. As the linear subsystem $\bar{G}_{i}(s)$ is asymptotically stable with unit static gain, it follows from (3)-(4a) that the internal signal $u_{i}(t)$ turns out to be (in steady state)

$$
\begin{equation*}
u_{i}(t)=x_{0}+V\left|\bar{G}_{i}(j \omega)\right| \sin \left(\omega t+\varphi_{i}(\omega)\right), \tag{16}
\end{equation*}
$$

with $\varphi_{i}(\omega)=\arg \left(\bar{G}_{i}(j \omega)\right)$. Also, it is readily obtained using (3), (14a-b) and (16)

$$
\begin{equation*}
u_{o}(t) \approx \sum_{l=0}^{p} \bar{c}_{l}\left(x_{0}+V\left|\bar{G}_{i}(j \omega)\right| \sin \left(\omega t+\varphi_{i}(\omega)\right)\right)^{l}, \tag{17}
\end{equation*}
$$

The factor multiplying $\bar{c}_{l}$ in (17) is written as

Specifically, one has for $j=1 \ldots n$

$$
\begin{align*}
&\left(x_{0}+V\left|\bar{G}_{i}(j \omega)\right| \sin \left(\omega t+\varphi_{i}(\omega)\right)\right)^{l}  \tag{18a}\\
&=\sum_{r=0}^{l} C_{r}^{l} x_{0}{ }^{\prime}\left(V\left|\bar{G}_{i}(j \omega)\right| \sin \left(\omega t+\varphi_{i}(\omega)\right)\right)^{l-r},
\end{align*}
$$

Using the standard trigonometric formulas, the right side of (26) simplifies to

$$
\begin{equation*}
y(t)=s_{0}+\sum_{k=1}^{\infty} s_{k} \sin \left(k \omega t+\phi_{k}\right)+\xi(t) \tag{27}
\end{equation*}
$$

with

$$
\begin{gather*}
s_{k}=\sqrt{a_{k}^{2}+b_{k}^{2}} \text { for } k=1,2, \ldots  \tag{28a}\\
\phi_{k}=\tan ^{-1}\left(\frac{a_{k}}{b_{k}}\right) \text { for } k=1,2, \ldots  \tag{28b}\\
s_{0}=\frac{a_{0}}{2}=\frac{1}{T} \int_{0}^{T} w(t) d t \tag{28c}
\end{gather*}
$$

Knowing that

$$
\begin{equation*}
\lim _{k \rightarrow \infty} s_{k}=0, \tag{28d}
\end{equation*}
$$

Remark 2. a) Considering the above assumptions, practically, it is reasonable to limit the development in Fourier series of $y(t)$ to those frequencies for which the Fourier series coefficients are significant. Furthermore, it readily follows from (26)-(28d) that $\sum_{k=p+1}^{\infty} \frac{s_{k}^{2}}{2}=0$ as $p \rightarrow \infty$. Then, it is reasonable to consider as significant only the coefficient list $s_{k}(k=0,1,2, \ldots, p)$.
b) The choice of the polynomial order $p$ is as follows:

For any point $x_{0}$ in the working interval and a given error $\varepsilon$ $(=1 \%, 2 \%, \ldots)$, let $P_{t}$ and $P_{p}$ denote, respectively, the total power and the power of the first $p$ components of the output signal. Using the Parseval's identity (e.g. Ljung, 1987)

$$
\begin{gather*}
P_{t}=\frac{a_{0}^{2}}{4}+\sum_{k=1}^{\infty} \frac{a_{k}^{2}+b_{k}^{2}}{2}=s_{0}^{2}+\sum_{k=1}^{\infty} \frac{s_{k}^{2}}{2}=\frac{1}{T} \int_{0}^{T} y^{2}(t) d t<\infty  \tag{29a}\\
P_{p}=\frac{a_{0}^{2}}{4}+\sum_{k=1}^{p} \frac{a_{k}^{2}+b_{k}^{2}}{2}=s_{0}^{2}+\sum_{k=1}^{p} \frac{s_{k}^{2}}{2}<\infty \tag{29b}
\end{gather*}
$$

Finally, for $p=1,2, \ldots$, we seek the minimum order $p$ satisfying the following condition

$$
\begin{equation*}
P_{p} \geq\left(\frac{100-\varepsilon}{100}\right) P_{t} \tag{30}
\end{equation*}
$$

24) for some $0<\varepsilon \ll 100 \%$ that is chosen by the user. The amplitude $V$ of the input signal is reduced if necessary.

If the condition (30) holds, then the right side of (27) simplifies to

$$
\begin{equation*}
y(t)=s_{0}+\sum_{k=1}^{p} s_{k} \sin \left(k \omega t+\phi_{k}\right)+\xi(t) \tag{31}
\end{equation*}
$$

By using (23) and (31), we deduce the following relations

$$
\begin{gather*}
B_{k}\left(\left|\bar{G}_{i}(j \omega)\right|\right)\left|\bar{G}_{o}(j k \omega)\right|=s_{k} \text { for } k=1 \ldots p,  \tag{32a}\\
\beta_{k}\left(\varphi_{i}(\omega),\left|\bar{G}_{i}(j \omega)\right|\right)+\varphi_{o}(k \omega)=\phi_{k} \text { for } k=1 \ldots p, \tag{32b}
\end{gather*}
$$

Accordingly, equations (23) and (31) show how to obtain the complex amplitudes $\bar{G}_{i}(j \omega)$ and $\bar{G}_{o}(j k \omega)$ using the two couples $\left(B_{k}, \beta_{k}\right)$ and $\left(s_{k}, \phi_{k}\right) \quad(k=0,1, \ldots, p)$. This is performed noticing that the right side of $(23)$ is nothing other than the Fourier series expansion of the output signal $y(t)$, up to noise $\xi(t)$. Consequently, the procedure to estimate the output $y(t)$ is as follows: First, we assume that the condition (30) holds, so the equation (31) is maintained. Next, given that all deterministic terms on the right side of (31) are periodic, with common period $T$, and $\xi(t)$ is a zero-mean ergodic white noise, the effect of the latter can be filtered considering the following trans-period averaging of the output

$$
\begin{equation*}
y_{f}(t, M)=\frac{1}{M} \sum_{i=1}^{M} y(t+(i-1) T), \quad 0 \leq t<T \tag{33}
\end{equation*}
$$

for some (large enough) integer $M$. Indeed, it is readily obtained using (31) and (33)

$$
\begin{align*}
\lim _{M \rightarrow \infty} y_{f}(t, M) & =s_{0}+\sum_{k=1}^{p} s_{k} \sin \left(k \omega t+\phi_{k}\right) \\
& +\lim _{M \rightarrow \infty} \frac{1}{M} \sum_{i=1}^{M} \xi(t+(i-1) T),  \tag{34}\\
& =s_{0}+\sum_{k=1}^{p} s_{k} \sin \left(k \omega t+\phi_{k}\right)
\end{align*}
$$

where the last equality holds with probability 1 because $\xi$ is ergodic and zero-mean. That is, the $s_{k}$ 's and $\phi_{k}$ 's turn out to be (w.p.1) the limits of Fourier series parameters of $y_{f}(t, M)$ as $M \rightarrow \infty$. These parameters are given by the usual expressions

$$
\begin{gather*}
s_{k}(M)=\sqrt{a_{k}(M)^{2}+b_{k}(M)^{2}} \quad(k=1,2, \ldots, p),  \tag{35a}\\
\phi_{k}(M)=\tan ^{-1}\left(\frac{a_{k}(M)}{b_{k}(M)}\right) \quad(k=1,2, \ldots, p),  \tag{35b}\\
s_{0}(M)=\frac{1}{T} \int_{0}^{T} y_{f}(t, M) d t=\frac{a_{0}(M)}{2}, \tag{35c}
\end{gather*}
$$

where

$$
\begin{align*}
& a_{k}(M)=\frac{2}{T} \int_{0}^{T} y_{f}(t, M) \cos (k \omega t) d t \\
& b_{k}(M)=\frac{2}{T} \int_{0}^{T} y_{f}(t, M) \sin (k \omega t) d t \tag{36}
\end{align*}
$$

Then, it follows from (34)-(36) that

$$
\begin{equation*}
\lim _{M \rightarrow \infty} s_{k}(M)=s_{k}(\text { w.p.1) for }(k=0 \ldots p) \tag{37a}
\end{equation*}
$$

$$
\begin{equation*}
\lim _{M \rightarrow \infty} \phi_{k}(M)=\phi_{k} \quad(\text { w.p.1) for }(k=1 \ldots p) \tag{37b}
\end{equation*}
$$

Accordingly, the above results lead to get the estimates $\hat{G}_{i}(j \omega, M)$ and $\hat{G}_{o}(j k \omega, M)$ of the respective complex gains $\bar{G}_{i}(j \omega)$ and $\bar{G}_{o}(j k \omega) \quad(k=1, \ldots, p)$, for any fixed $\omega>0$. This is performed using the identification procedure of Table 2.

Proposition 2. The estimates $\hat{G}_{i}(j \omega, M)$ and $\hat{G}_{o}(j k \omega, M)$ obtained by the FGI procedure are consistent i.e. $\hat{G}_{i}(j \omega, M) \rightarrow \bar{G}_{i}(j \omega)$ and $\quad \hat{G}_{o}(j k \omega, M) \rightarrow \bar{G}_{o}(j k \omega) \quad$ w.p. 1 as $M \rightarrow \infty$.

Proof. Dice that (39) is verified, furthermore using results of (34)-(37b) and making use of (41a-b), it is readily seen that the estimates $\hat{G}_{i}(j \omega, M)$ and $\hat{G}_{o}(j k \omega, M)$ converge in probability, respectively, to $\bar{G}_{i}(j \omega)$ and $\bar{G}_{o}(j k \omega)$.

Remark 4. For a single experiment (a single frequency $\omega), p$ estimates of the complex gain $\bar{G}_{o}(j \omega)$ are obtained (i.e. $\left|\hat{G}_{o}(j k \omega, M)\right| \quad$ and $\quad \hat{\varphi}_{o}(j k \omega, M) \quad$ for $\left.\quad k=1,2, \ldots, p\right)$ in addition to the estimate of $\bar{G}_{i}(j \omega)$.

## V. SimULATION

The system to be identified is analytically described by equations (1)-(3) with

$$
\begin{gather*}
G_{i}(s)=\frac{0.01}{(s+0.1)(s+0.5)}, \quad G_{o}(s)=\frac{0.1}{(s+0.2)(s+0.01)},  \tag{42aa}\\
f(x)=\exp (x) . \tag{42b}
\end{gather*}
$$

Then, the parameterized system will be identified

$$
\begin{gather*}
\bar{G}_{i}(s)=\frac{0.05}{(s+0.1)(s+0.5)}, \bar{G}_{o}(s)=\frac{0.002}{(s+0.2)(s+0.01)}  \tag{43a}\\
\bar{f}(x)=G_{o}(0) f\left(G_{i}(0) x\right)=50 \exp (0.2 x) \tag{43b}
\end{gather*}
$$

$\xi(t)$ is a sequence of uniform random numbers in $\left[\begin{array}{cc}-1 & 1\end{array}\right]$. The above system is submitted to piecewise constant input (Fig. 2). Fig. 2 also shows the output signal. Fig. 3 shows the nonlinearity $\bar{f}($.$) considered in simulation$ and the couples estimates $\left(V_{j}, \hat{W}_{j}(N)\right)(j=1 \ldots 7)$ for $N=10$.

TABLE II. Frequency Gains Identification (FGI)

| 1. Data acquisition |
| :--- |
| - Choose a frequency $\omega \in\left\{\omega_{1}, \omega_{2}, \ldots, \omega_{N}\right\}$, let $T=2 \pi / \omega$ |
| - Apply the sine signal, with period $T$, defined as follows: |
| $\qquad v(t)=x_{0}+V \sin (\omega t) \quad t \geq 0$ |
| where $x_{0} \in\left[v_{m} v_{M}\right]$ is arbitrarily chosen. |
| - Take a sufficiently long record of the resulting steady-state |
| output signal. Let it be denoted $y(t), t \in[0 M T]$ for some |
| integer $M \gg 1$. |
| 2. Estimate of the filtered output |


| 1. Data acquisition |
| :---: |
| Fix a given error $\varepsilon(1 \%, 2 \%, \ldots)$, select an order $p$ | satisfying the following condition

$$
\begin{equation*}
P_{p} \geq\left(\frac{100-\varepsilon}{100}\right) P_{t} \tag{39}
\end{equation*}
$$

where $P_{t}$ is the total power output signal and $P_{p}$ is the power of the first $p$ harmonics of $y(t)$.

- If necessary, vary the point $x_{0}$ and reduce the amplitude $V$.
- Generate the filtered output $y_{f}(t, M)$ using (33) and compute its Fourier series coefficients $a_{k}(M)$ and $b_{k}(M)$ using (36). Then, deduce the parameters $S_{k}(M)$ and $\phi_{k}(M)$ using (35a-c).


## 3. Data processing

- From the input sequence $v(t)$, using the equations (14a)(22) and accompanying remarks, develop the Fourier series decomposition of the undisturbed output $w(t)$ according to the parameters of linear subsystems; the latter may have the following expression

$$
\begin{equation*}
w(t)=\sum_{k=0}^{p} B_{k}\left|\bar{G}_{o}(j k \omega)\right| \sin \left(k \omega t+\beta_{k}+\varphi_{o}(k \omega)\right) \tag{40}
\end{equation*}
$$

where the parameter $B_{k}$ (resp. $\beta_{k}$ ), for $k=1 \ldots p$, depends on $\left|\bar{G}_{i}(j \omega)\right|$ (resp. $\varphi_{i}(\omega)$ and $\left|\bar{G}_{i}(j \omega)\right|$ ) for fixed $V$ and $x_{0}$.

- Compute the estimates $\hat{G}_{i}(j \omega, M)$ and $\hat{G}_{o}(j k \omega, M)$ using the filtered output $y_{f}(t, M)$ and (40), i.e. for $k=1, \ldots, p$

$$
\begin{gather*}
B_{k}\left(\left|\bar{G}_{i}(j \omega)\right|\right)\left|\bar{G}_{o}(j k \omega)\right|=s_{k}(M)  \tag{41a}\\
\beta_{k}\left(\varphi_{i}(\omega),\left|\bar{G}_{i}(j \omega)\right|\right)+\varphi_{o}(k \omega)=\phi_{k}(M) \tag{41b}
\end{gather*}
$$

4. Estimation for all frequencies

- Repeat steps 1 through 3 for all frequencies $\left\{\omega_{1}, \ldots, \omega_{N}\right\}$.


Figure 2. Shape of the resulting disturbed output signal.


Figure 3. Nonlinear element $\bar{f}($.$) and \left(V_{j}, \hat{W}_{j}(N)\right)$.

The system is now excited by the sine signal (38). For an error $\varepsilon=0.6 \%$ and $V \leq 1.5$, the condition (39) is satisfied if $p \geq 3$. Let fix $p=3$. Then, it follows from the power formulas (20a-b), the standard trigonometric formulas and using the procedure as explained in Section 4, one immediately gets

$$
\begin{align*}
s_{0} & =\bar{c}_{0}+\frac{1}{2} \bar{c}_{2}\left(V\left|\bar{G}_{i}(j \omega)\right|\right)^{2} \\
s_{1} & =V\left|\bar{G}_{i}(j \omega)\right|\left|\bar{G}_{o}(j \omega)\right|\left(\bar{c}_{1}+\frac{3}{4} \bar{c}_{3}\left(V\left|\bar{G}_{i}(j \omega)\right|\right)^{2}\right),  \tag{44a}\\
s_{2} & =-\frac{1}{2} \bar{c}_{2}\left(V\left|\bar{G}_{i}(j \omega)\right|\right)^{2}\left|\bar{G}_{o}(j 2 \omega)\right| \\
s_{3} & =-\frac{1}{4} \bar{c}_{3}\left(V\left|\bar{G}_{i}(j \omega)\right|\right)^{3}\left|\bar{G}_{o}(j 3 \omega)\right| \\
\phi_{1}(\omega) & =\varphi_{i}(\omega)+\varphi_{o}(\omega) ;  \tag{44b}\\
\phi_{2}(\omega) & =2 \varphi_{i}(\omega)+\varphi_{o}(2 \omega)+\frac{\pi}{2} ; \phi_{3}(\omega)=3 \varphi_{i}(\omega)+\varphi_{o}(3 \omega)
\end{align*}
$$

The coefficients $\bar{c}_{l}(l=0, \ldots, 3)$ can be estimated using the NI procedure for 4 values $V_{j} \leq 1.5 \quad(j=1, \ldots, 4)$

$$
\begin{equation*}
\hat{\bar{c}}_{0}=49.7 ; \quad \hat{\bar{c}}_{1}=10.1 ; \quad \hat{\bar{c}}_{2}=0.97 ; \quad \hat{\bar{c}}_{3}=0.075 \tag{45}
\end{equation*}
$$

The Fourier coefficients $s_{k}$ and $\phi_{k}$ can be determined by using the filtered output (33) and the equations (35a)-(36). Then the gains $\left|\bar{G}_{i}(j \omega)\right|$ and $\left|\bar{G}_{o}(j k \omega)\right|$ (with $k=0, \ldots, 3$ ) can be estimated directly from (44a). The estimation of phases is performed by exciting with frequencies equal to $2 \omega$ and $3 \omega$ . Finally, for $\omega=0.01 r d / s$ and $M=40$, the following results are obtained
$\left|\hat{G}_{i}(j \omega, M)\right|=1.03 ;\left|\hat{G}_{o}(j \omega, M)\right|=0.74 ;\left|\hat{G}_{o}(j 2 \omega, M)\right|=0.41 ;$
$\left|\hat{G}_{o}(j 3 \omega, M)\right|=0.29 ; \hat{\varphi}_{i}(j \omega, M)=-0.11 ; \hat{\varphi}_{o}(j \omega, M)=-0.85$;
$\hat{\varphi}_{i}(j 2 \omega, M)=-0.27 ; \hat{\varphi}_{o}(j 2 \omega, M)=-1.18 ; \hat{\varphi}_{i}(j 3 \omega, M)=-0.33 ;$,
$\hat{\varphi}_{o}(j 3 \omega, M)=-1.42 ; \hat{\varphi}_{o}(j 4 \omega, M)=-1.55 ; \hat{\varphi}_{o}(j 6 \omega, M)=-1.64 ;$
$\hat{\varphi}_{o}(j 9 \omega, M)=-1.85$

Remark 3. During the identification of the phase of the linear subsystems, the number of unknown parameters is sometimes greater than the number of equations, to solve this problem the FGI procedure (Table 2) is repeated for multiples of frequency $\omega$ (i.e. $2 \omega, \ldots, p \omega$ ).

## VI. Conclusion

We have developed a new two-stage frequency identification method to deal with WH systems identification. The originality of the present study lies in the fact that the phases of the two linear subsystems can be separated, also both linear subsystems are nonparametric and of unknown structure. Accordingly, the linear subsystems are not necessarily finite order. The nonlinear element $f($.) has any form and, in particular, may be noninvertible. This is only supposed to be accurately represented, within any finite interval, with a polynomial of finite order. Another feature of the method is the fact that the excitation signals are easily generated and the estimation algorithms can be simply implemented, compared with several published approaches. Finally, we note that the choice of the frequency band of interest is not required.

## References

[1] E.W. Bai, Z. Cai, S. Dudley-Javorosk, R.K. and Shields, "Identification of a modified Wiener-Hammerstein system and its application in electrically stimulated paralyzed skeletal muscle modelling," Automat. Contr., vol. 45, pp. 736-743, 2009.
[2] A. Marconato, J. Sjoberg, and J. Schoukens, "Initialization of nonlinear state-space models applied to the Wiener-Hammerstein benchmark," Control Engineering Practice, vol. 20, pp. 1126-1132, 2012.
[3] N.J. Bershad, P. Celka, and S. McLaughlin, "Analysis of stochastic gradient identification of Wiener-Hammerstein systems for nonlinearities with Hermite polynomial expansions," IEEE Trans. Signal Process., vol. 49, pp. 1060-1071, 2001.
[4] G. Pillonetto, A. Chiuso, and G. De Nicolao, "Prediction error identification of linear systems: a nonparametric Gaussian regression approach," Automatica, vol. 47, no. 2, pp. 291-305, 2011.
[5] F. Giri, Y. Rochdi, A. Radouane, A. Brouri, F.Z. Chaoui, "Frequency Identification of nonparametric Wiener systems containing backlash nonlinearities," Automatica, vol. 49, pp. 124-137, 2013.
[6] A. Brouri, F. Giri, F. Ikhouane, F.Z. Chaoui, and O. Amdouri, "Identification of Hammerstein- Wiener systems with Backlash input nonlinearity bordered by straight lines," in 19th IFAC, Cape Town, 2014, pp. 475-480.
[7] J. Paduart, L. Lauwers, R. Pintelon, and J. Schoukens, "Identification of a Wiener-Hammerstein system using the polynomial nonlinear state space approach," Control Engineering Practice, vol. 20, pp. 1133-1139, 2012.
[8] L. Ljung, "System Identification - Theory for the User," PrenticeHall, Englewood Cliffs, N J, 1987.

Adil BROURI. In 2000, he obtained the Agrégation of Electrical Engineering from the ENSET, Rabat, Morocco and, in 2012, he obtained a Ph.D. in Automatic Control from the University of Mohammed 5, Morocco. He has been Professeur-Agrégé for several years. Since 2013 he joined the ENSAM, My Ismail University in Meknes-Morocco. He obtained his HDR degree in 2015 at the ENSAM-Meknes (My Ismail University). His research interests include nonlinear system identification and nonlinear control. He published several papers on these topics. Pr. Adil BROURI joined the research team "IMSM" and the L2MC Laboratory as a permanent member.


[^0]:    * ENSAM, L2MC, Moulay Ismail University, Morocco.

    Adil Brouri is with the ENSAM, L2MC, AEEE Department, Moulay Ismail University, Meknes, Morocco (e-mail: a.brouri@ensam-umi.ac.ma \& brouri_adil@ yahoo.fr).

