

Analysis of Modal Insensitivity for Generator Using Eigenstructure Assignment

Shu-Chen Wang

Abstract—This paper studies the modal insensitivity design of the synchronous generator in a power system using eigenstructure assignment. Modal insensitivity means that the concerned system mode shape is insensitive to small variations in the system. In the proposed design procedure, modal insensitivity is achieved by assigning system eigenstructure and only algebraic computations are involved. Results from the study of a single-machine-infinite-bus system are presented.

Keywords—Power system control, synchronous generator, eigenvalue, eigenstructure assignment, modal insensitivity.

I. INTRODUCTION

Eigenstructure assignment refers to the design method for assigning the eigenstructure, i.e. eigenvalues and eigenvectors, of a closed-loop linear system with a constant gain feedback control law [1-22]. Methods of eigenstructure assignment have been proposed and applied to various kinds of control design problems. That includes some applications in power system control [23-44].

The design of linear control system is based on the linearized system model. However, physical system are often subject to operating point drift or plant parameter variations which may cause that the designed controller does not meet the prescribed performance, even result in system instability. Modal insensitivity means that system mode shape is insensitive to small variations [18-23]. Controllers designed with the concept of modal insensitivity will have insensitive closed-loop eigenvalue or eigenvectors.

Based on eigenstructure assignment techniques, the main purpose of this paper is to present a modal insensitivity design for control of synchronous generator in a power system with a view to making the eigenvalues of main system oscillatory modes insensitive to the variation of a certain parameter. The point is to design an output feedback controller for assigning the system eigenvalues as well as for restricting some specified eigenvector components. The design procedure in this paper needs only algebraic computations and thus is easy to apply.

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II. DESIGN METHODOLOGY

A. Eigenstructure Assignment

Consider a dynamic system described by (1) and (2).

$$\dot{x} = Ax + Bu \quad (1)$$

$$y = Cx \quad (2)$$

Assume that the system has n states, m inputs and r outputs. An output feedback control $u = Gy = GCx$ will result in the closed-loop system

$$\dot{x} = (A + BGC)x \quad (3)$$

Denote the corresponding right eigenvectors of the i -th eigenvalue λ_i as V_i and assume the following definitions:

$$S_{\lambda_i} \triangleq [\lambda_i I - A \quad -B] \quad (4)$$

$$\text{Ker}[S_{\lambda_i}] \triangleq \begin{bmatrix} N_{\lambda_i} \\ M_{\lambda_i} \end{bmatrix} \quad (5)$$

$$W_i \triangleq GC V_i \quad (6)$$

The following theorem for eigenstructure assignment has been established [1-22].

Theorem 1 Given a controllable and observable system described by (1) and (2). Let $\{\lambda_i\}_{i=1}^n$ and $\{v_i\}_{i=1}^n$ be the self-conjugate sets of desired eigenvalues/eigenvectors, respectively. There exists an $m \times r$ real matrix G such that $(A + BGC)V_i = \lambda_i V_i$ if and only if all the following conditions are satisfied:

- 1) $\{V_i\}_{i=1}^n$ is a linearly independent set on C^n ;
- 2) $V_i = V_j^*$ if $\lambda_i = \lambda_j^*$; (7)

- 3) $\begin{bmatrix} V_i \\ W_i \end{bmatrix} \in \text{span} \begin{bmatrix} N_{\lambda_i} \\ M_{\lambda_i} \end{bmatrix}$; (8)

$$4) \text{rank} \begin{bmatrix} CV_1 & CV_2 & \dots & CV_n \\ W_1 & W_2 & \dots & W_n \end{bmatrix} = r \quad (9)$$

B. Modal Insensitivity

Consider a system with the following dynamic equations

$$\dot{x} = A(\alpha)x + B(\alpha)u \quad (10)$$

$$y = C(\alpha)x \quad (11)$$

Note that the matrices $A(\alpha)$, $B(\alpha)$ and $C(\alpha)$ depend on the parameter α whose original value is α_0 .

If we apply an output feedback control law $u = G y = GC(\alpha)x$ then the closed-loop system becomes

$$\dot{x} = [A(\alpha) + B(\alpha)GC(\alpha)]x \equiv \bar{A}(\alpha)x \quad (12)$$

We are to design an output feedback gain matrix G such that $\bar{A}(\alpha_0)$ has the prescribed eigenvalues that are insensitive to small variation in α around α_0 .

Denote the corresponding left and right eigenvectors of the i -th eigenvalue λ_i as U_i and V_i , respectively. The variation of closed-loop system matrix, $d\bar{A}$, will be

$$\begin{aligned} d\bar{A} &= \left[\frac{dA}{d\alpha} \Big|_{\alpha_0} + \frac{dB}{d\alpha} \Big|_{\alpha_0} GC(\alpha_0) + B(\alpha_0)G \frac{dC}{d\alpha} \Big|_{\alpha_0} \right] d\alpha \\ &\equiv [\delta A + \delta BGC + BG\delta C]d\delta \\ &\equiv [\delta \bar{A}]d\delta \end{aligned} \quad (13)$$

Then the following theorem can be obtained.

Theorem 2 The necessary and sufficient conditions for λ_i and U_i (or V_i) to be insensitive to a small variation in α are:

$$\begin{bmatrix} U_i^T & U_i^T BK \end{bmatrix} \begin{bmatrix} \delta A & \delta B \\ \delta C & 0 \end{bmatrix} = 0 \quad (14)$$

$$\begin{bmatrix} \delta A & \delta B \\ \delta C & 0 \end{bmatrix} \begin{bmatrix} V_i \\ KCV_i \end{bmatrix} = 0 \quad (15)$$

C. Design Procedure

Theorem 1 and Theorem 2 form the basis of the control design approach in this paper. The basic idea is to fully exploit the flexibility offered by feedback beyond closed-loop eigenvalue/eigenvector, i.e. eigenstructure, assignment for restricting certain eigenvector elements in order to achieve modal insensitivity.

The following steps summarize the design procedure for mode-insensitive control:

- 1) Apply Theorem 1 to deriving algebraic equations with unknowns being the elements in the output feedback gain matrix. The emphasis here is placed on assigning eigenvalues to improve system damping.
- 2) Utilize Theorem 2 to solve the algebraic equations derived in Step 1 for finding the exact solution of the output feedback gain matrix. The resultant closed-loop eigenvalues will have the property of being insensitive.
- 3) Verify the control design by calculating the eigenvalue sensitivity and closed-loop system eigenvalues.

Note that the design procedure needs only algebraic manipulations and exact solution can be obtained without any kind of iteration.

III. STUDY SYSTEM

The system considered in this work is a synchronous generator connected to a large power system whose linearized model is shown in Fig.1. This linear model has been extensively studied in the literature of power system control. The dynamic characteristics of the system are expressed in terms of the six constants $K_1 - K_6$. The studied system can then be represented by the state space form in (1) and (2).

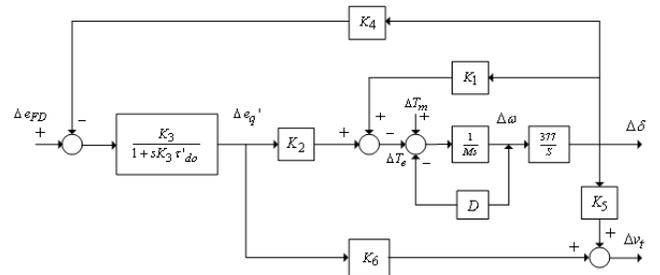


Fig. 1 Linearized model of single-machine-infinite-bus

The definitions for each variable are

e'_q Voltage proportional to direct axis flux linkage

ω Rotor speed

δ Torque angle

e_{FD} Generator field voltage

T_m Mechanical torque

where $x = [\Delta e'_q \ \Delta \omega \ \Delta \delta]^T$, $u = [\Delta e_{FD} \ \Delta T_m]^T$ and $y = [\Delta \omega \ \Delta \delta]^T$ are the state, control and output vectors, respectively.

The matrices A , B , and C in (1) and (2) are

$$A = \begin{bmatrix} -1/(K_3\tau'_{do}) & 0 & -K_4/\tau'_{do} \\ -K_2/M & -D/M & -K_1/M \\ 0 & \omega_R & 0 \end{bmatrix} \quad (16)$$

$$B = \begin{bmatrix} -1/\tau'_{do} & 0 \\ 0 & 1/M \\ 0 & 0 \end{bmatrix} \quad (17)$$

$$C = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad (18)$$

Note that the system parameter K_2 is the change in electrical torque for small change in the d-axis flux linkage at constant rotor angle as described in (19)

$$K_2 = \left. \frac{\Delta T_e}{\Delta e'_q} \right|_{\delta = \delta_0} \quad (19)$$

If the field flux linkage is constant, then e'_q will be constant and $K_2 = 0$, and the model is reduced to the classical model. The effect of K_2 variation on the electromechanical mode eigenvalues can be investigated by computing the sensitivity of any eigenvalue of interest λ with respect to the parameter K_2 , i.e. $\partial \text{Re}(\lambda)/\partial K_2$. The task here is to design a controller to make the electromechanical mode eigenvalues to be insensitive to K_2 .

IV. EXAMPLE

Consider the linearized model of a synchronous generator connected to a large power system as shown in Fig.1. The parameters tabulated in Table 1 are taken for this study.

TABLE I
PARAMETERS FOR STUDY SYSTEM

$K_1 = 1.0755$	$K_2 = 1.2578$	$K_3 = 0.3072$
$K_4 = 1.7124$	$K_5 = -0.0409$	$K_6 = 0.4971$
$\tau'_{do} = 5.9$	$M = 4.74$	$D = 0$

Then the numerical values for matrices A and B of the system model in (16) and (17) are obtained as

$$A = \begin{bmatrix} -0.5517 & 0 & -0.2909 \\ -0.2654 & 0 & -0.2269 \\ 0 & 377 & 0 \end{bmatrix} \quad (20)$$

$$B = \begin{bmatrix} 0.1695 & 0 \\ 0 & 0.211 \\ 0 & 0 \end{bmatrix} \quad (21)$$

The open-loop eigenvalues and left/right eigenvectors for the study system are calculated as (22) and (23), respectively.

$$\begin{aligned} \lambda_1 &= -0.2124, \\ \lambda_2 &= -0.1696 + j9.2434, \\ \lambda_3 &= -0.1696 - j9.2434 \end{aligned} \quad (22)$$

$$\begin{aligned} U_1 &= \begin{bmatrix} 1.5410 \\ -1.9699 \\ 0.0011 \end{bmatrix}, \\ U_2 &= \begin{bmatrix} 0.2893 + j0.5096 \\ 17.3325 - j10.8083 \\ 0.2572 + j0.4298 \end{bmatrix}, \\ U_3 &= \begin{bmatrix} 0.2893 - j0.5096 \\ 17.3325 + j10.8083 \\ 0.2572 - j0.4298 \end{bmatrix}, \end{aligned} \quad (23)$$

$$\begin{aligned} V_1 &= \begin{bmatrix} 0.6500 \\ 0.0004 \\ -0.7599 \end{bmatrix}, \\ V_2 &= \begin{bmatrix} 0.0265 + j0.0167 \\ 0.0210 + j0.0126 \\ 0.4973 - j0.8667 \end{bmatrix}, \\ V_3 &= \begin{bmatrix} 0.0265 - j0.0167 \\ 0.0210 - j0.0126 \\ 0.4973 + j0.8667 \end{bmatrix}. \end{aligned}$$

The pair of eigenvalues λ_2, λ_3 , often referred to as the electromechanical mode, is the eigenvalues associated with generator rotor oscillation. It is the primary objective of this study to design an output feedback controller to enhance the damping of the electromechanical mode. We can analyze the effect of K_2 variation on the electromechanical mode eigenvalues by computing the following sensitivity:

$$\begin{aligned} \frac{\partial \lambda_1}{\partial K_2} &= 0.269, \\ \frac{\partial \text{Re}(\lambda_2)}{\partial K_2} &= \frac{\partial \text{Re}(\lambda_3)}{\partial K_2} = -0.134, \\ \frac{\partial \text{Im}(\lambda_2)}{\partial K_2} &= \frac{\partial \text{Im}(\lambda_3)}{\partial K_2} = \pm 6.3 \times 10^{-4} \end{aligned} \quad (24)$$

From (24), it is obvious that the electromechanical mode eigenvalues are much affected by K_2 . It is desirable that the assigned closed-loop eigenvalues be insensitive to K_2 .

Now the closed-loop eigenvalues are chosen to be

$$\begin{aligned}\lambda_1 &= -0.5517, \\ \lambda_2 &= -1.0 + j9.2434, \\ \lambda_3 &= -1.0 - j9.2434\end{aligned}\quad (25)$$

and the open-loop left and right eigenvectors are calculated as (26), respectively.

$$\begin{aligned}U^T &= \begin{bmatrix} 1.5378 & 0 & 0 \\ 0.4741 - j0.3428 & -11.1387 - j17.0896 & 0.3895 - j0.3184 \\ 0.4741 + j0.3428 & -11.1387 + j17.0896 & 0.3895 + j0.3184 \end{bmatrix}, \\ V &= \begin{bmatrix} 0.6503 & 0 & 0 \\ 0.0011 & -0.0156 + j0.0191 & -0.0156 - j0.0191 \\ -0.7597 & 0.8375 + j0.5459 & 0.8375 - j0.5459 \end{bmatrix}\end{aligned}\quad (26)$$

Note that a real part of -1.0 for electromechanical mode eigenvalues will give satisfactory dynamic behavior and the oscillation frequency is kept to remain unaltered. It is required that the assigned closed-loop eigenvalues be insensitive to parameter K_2 .

Since the system output variables are $\Delta\omega$ and $\Delta\delta$ we can express the output feedback control law as

$$\begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = \begin{bmatrix} g_{11} & g_{12} \\ g_{21} & g_{22} \end{bmatrix} \begin{bmatrix} y_1 \\ y_2 \end{bmatrix}\quad (27)$$

For the system specification mentioned above, the output feedback gain matrix G are obtained as

$$G = \begin{bmatrix} g_{11} & g_{12} \\ g_{21} & g_{22} \end{bmatrix} = \begin{bmatrix} 0 & 1.7121 \\ -9.4787 & -0.0113 \end{bmatrix}\quad (28)$$

and the closed-loop system matrix \bar{A} is

$$\bar{A} = (A + BGC) = \begin{bmatrix} -0.5517 & 0 & 0 \\ -0.2654 & -2 & -0.2293 \\ 0 & 377 & 0 \end{bmatrix}\quad (29)$$

For the purpose of verification, the eigenvalues of the closed-loop system matrix \bar{A} are calculated and found to be exactly as those in (25). Moreover, the sensitivity for each of the closed-loop system eigenvalues with respect to K_2 is computed as (30)

$$\frac{\partial\lambda_1}{\partial K_2} = \frac{\partial\lambda_2}{\partial K_2} = \frac{\partial\lambda_3}{\partial K_2} = 0\quad (30)$$

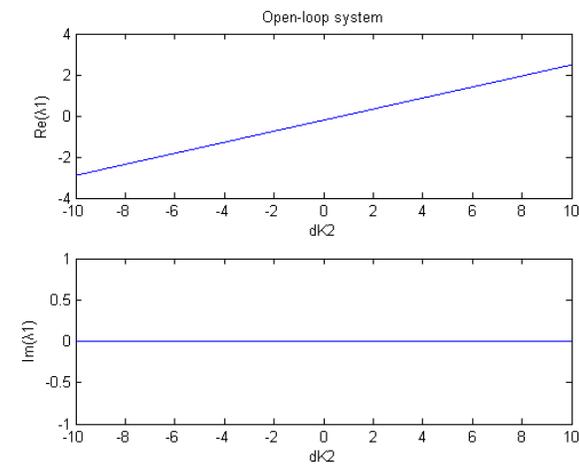
The eigenvalues in (25) with the values of sensitivity in (30) reveal that, with the designed output feedback controller, system damping has been greatly improved and the closed-loop system eigenvalues remain unaffected under the variations in parameter K_2 .

Simulation results are shown from Fig 2 to Fig 6. The control of the synchronous generator in a power system using modal insensitivity design is thus verified. Fig 2 and Fig 3 show the sensitivity of the open-loop system and closed-loop system eigenvalues under the variations in parameter K_2 , respectively. As compared to those in Fig 2 and Fig 3, the results in Fig 4 show that the proposed controller has improved on the sensitivity.

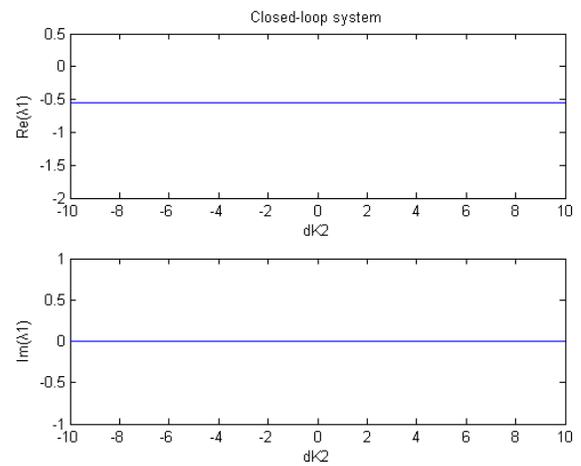
Dynamic responses of generator angle and speed for the open-loop system and closed-loop system under the variations in parameter K_2 are shown in Fig 5 and Fig 6, respectively.

V. CONCLUSION

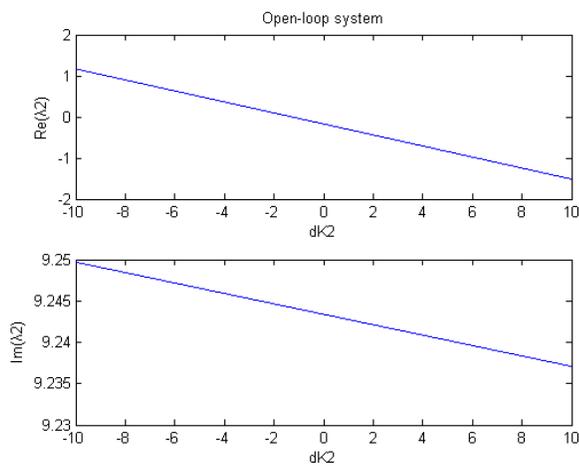
The use of modal insensitivity design for synchronous generator stabilization control in a power system has been presented in this paper. The design method is based on the concepts of eigenstructure assignment and sensitivity analysis. Only algebraic computations are needed and exact solution can be obtained without iteration. From the presented simulation results, it is found that the system damping has been greatly improved with the assigned eigenvalues, which are also made insensitive to the variations of the investigated parameter.



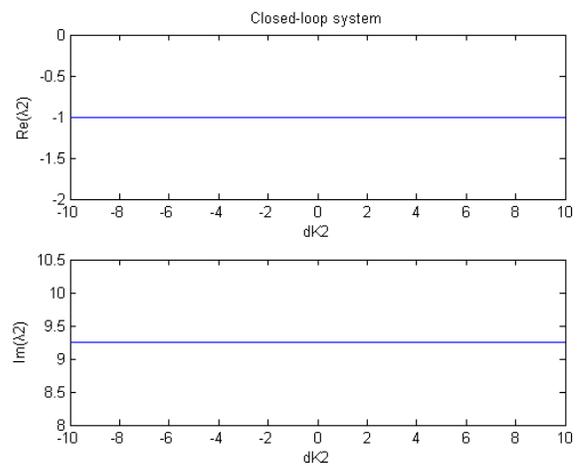
(a) eigenvalue λ_1



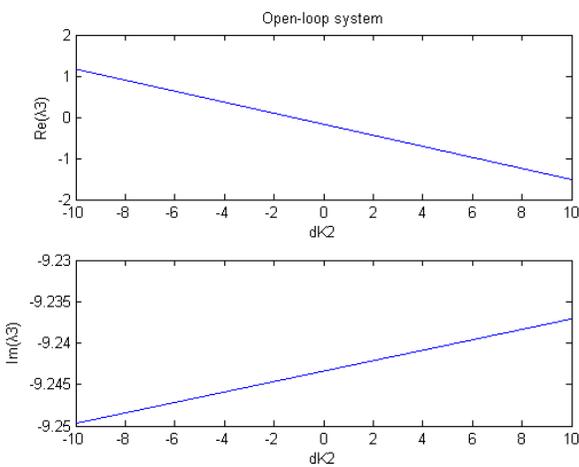
(a) eigenvalue λ_1



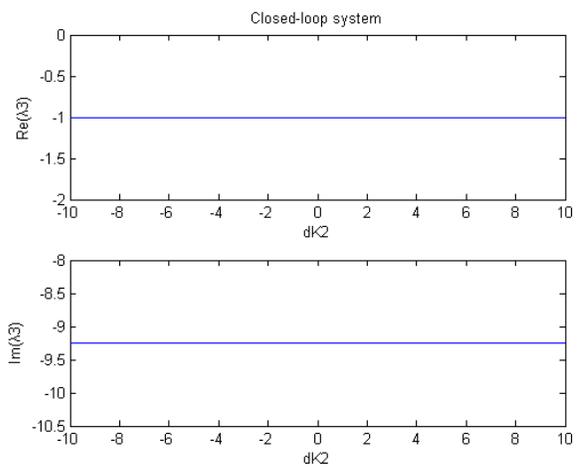
(b) eigenvalue λ_2



(b) eigenvalue λ_2



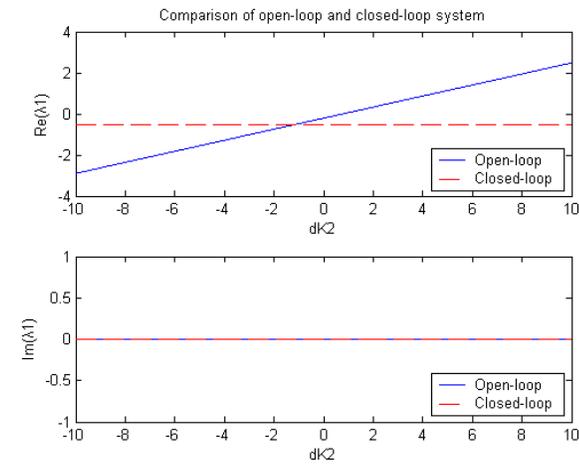
(c) eigenvalue λ_3



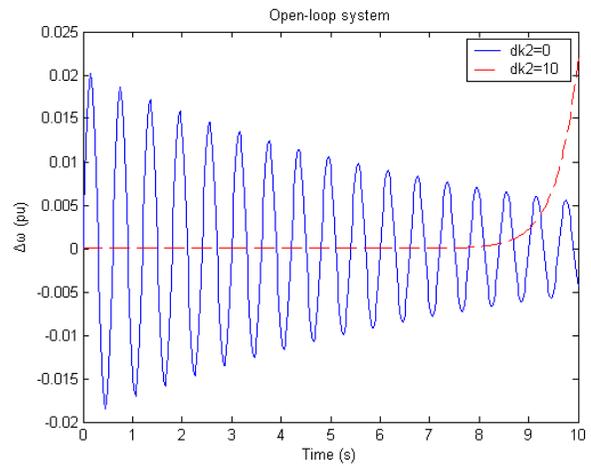
(c) eigenvalue λ_3

Fig. 2 The sensitivity of the open-loop system eigenvalues under the variations in parameter K_2

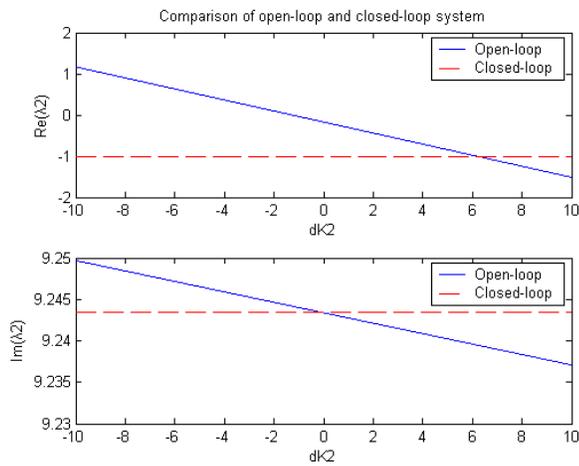
Fig. 3 The sensitivity of the closed-loop system eigenvalues under the variations in parameter K_2



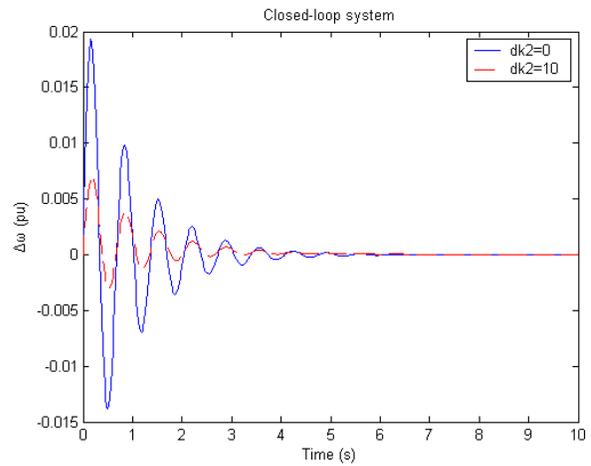
(a) eigenvalue λ_1



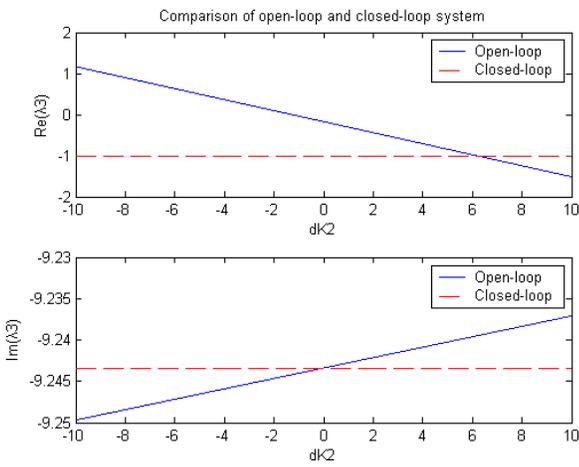
(a) Open-loop



(b) eigenvalue λ_2



(b) Closed-loop



(c) eigenvalue λ_3

Fig. 5 Dynamic responses of generator speed for the open-loop system and closed-loop system under the variations in parameter K_2

Fig. 4 Comparison of the sensitivity for the open-loop system and closed-loop system eigenvalues under the variations in parameter K_2

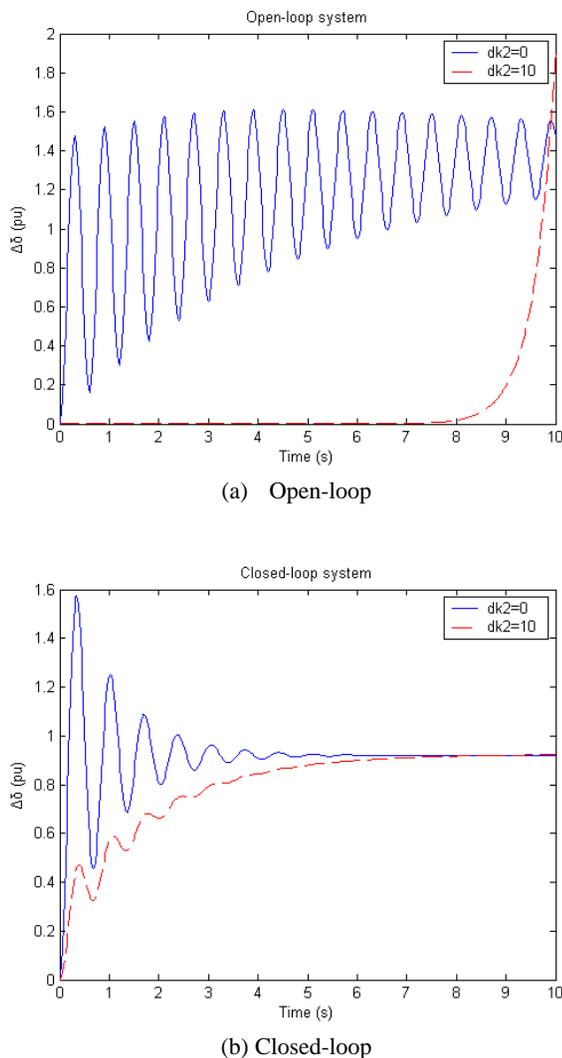


Fig. 6 Dynamic responses of generator angle for the open-loop system and closed-loop system under the variations in parameter K_2

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